comparison, the theoretical elastic breakdown pressure factor, based on the von Mises and Tresca yield criteria for the open end condition, are also shown. Based on the von Mises yield criterion and assuming  $\sigma_z = 0$ , elastic

$$PF = \frac{W^2 - 1}{\sqrt{(3W^4 + 1)}}\tag{1}$$

From the Tresea yield criterion elastic breakdown is:

$$PF = \frac{\mathbb{H}^{2} - 1}{2 \, \mathbb{H}^{2}} \tag{2}$$

As can be seen from the figure, there is close correlation between the experimentally determined and the theoretical von Mises elastic breakdown condition.

When the internal pressure exceeds the elastic breakdown pressure, the elastic-plastic interface moves from the bore towards the outside diameter. This movement is a function of the internal pressure, yield strength, diameter

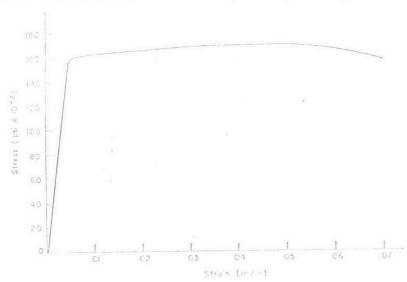


Fig. 8. Stress-strain diagram.

ratio and the strain hardening capabilities of the material. The strain hardening of this material at the yield-strength level considered is small, as shown in Fig. 8 which is a tensile stress-strain diagram for the material used in this program. As can be seen from the figure, for purposes of calculation and interpretation of the experimental results, the assumption of a elastic-perfectly

plastic material is valid.

The exterior surface of an open end cylinder subjected to internal pressure is in a condition of uniaxial stress since  $\sigma_r$  and  $\sigma_z$  are zero. Therefore, the condition of 100 percent overstrain may be defined as that at which the outside surface strain equals the strain associated with the yield stress of the material in uniaxial tension or

$$\epsilon_{E} = \frac{\sigma_y}{E}$$

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The pressure required to produce this condition  $(P_0)$  was experimentall determined for the specimens tested at should be noted that, since most of the special ass were designed for govern than 100 percent overstrain, there so contact between the specimen and container at the 100 percent andition. These values of  $P_0$  were converted to pressure factor, and all a suss for the same diameter ratio were averaged and plotted in Fig. 7. These data may be represented by an empirical relationship:

$$P_0 = 1.08 \sigma_y \log W \tag{4}$$

Weigle<sup>1</sup> gives the following equation for the 100 percent overstrain pressure based on the von Mises yield criterion and assuming  $\sigma_z = 0$ :

$$\begin{split} 2\sqrt{(3)} \tan^{-1} & |(\partial_0 - 1)^4| + \ln \partial_0 - \ln[1 + \sqrt{(3)}(\ell_0 - 1)^4] 2 \\ & + \ln 3 \ln^4 - \sqrt{(3)} \pi = 0 \end{split}$$

where

$$\partial_0 = \frac{4K^2}{P_0^2}$$

and K =yield stress in simple shear.

For ease of application, this equation may be approximated very accurately by the rectionship

$$P_0 = 1.10 \, e^{-1.10} \, W_1$$
 (6)

(6) refers to the diameter ratio It should be noted, however, that H' under pressure which is slightly less that mitial diameter ratio used in Eq. (4). Both equations, then, are in very a agreement and, for calculation purposes, the initial diameter ratio and (1) will be utilized.

The close agreement of Eq. (6) with the experimental data of Fig. 7 again verifies the assumption of an open end test condition.

## Partial Overstrain

In deriving relationships for stresses and strains in a partially overstrained cylinder the following basic assumptions are made.